



Welcome!

Webinar #16. GTTRAN: Transient Modelling

November 9, 2017

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Presenter: Evgeny Zakharenkov

Thermoflow Training and Support

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- On-site Training course
- Advanced Workshop
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- Technical Support

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Agenda

- Introduction to GTTRAN
- Steady state vs Transient modelling
- GTTRAN inputs and outputs
- Model sample
- Q & A session

Introduction to GTTRAN

- Transient (dynamic) boiler modelling software
- A separate standalone program licensed along with GT MASTER
- Added to GT MASTER for Version 23 (in 2013)

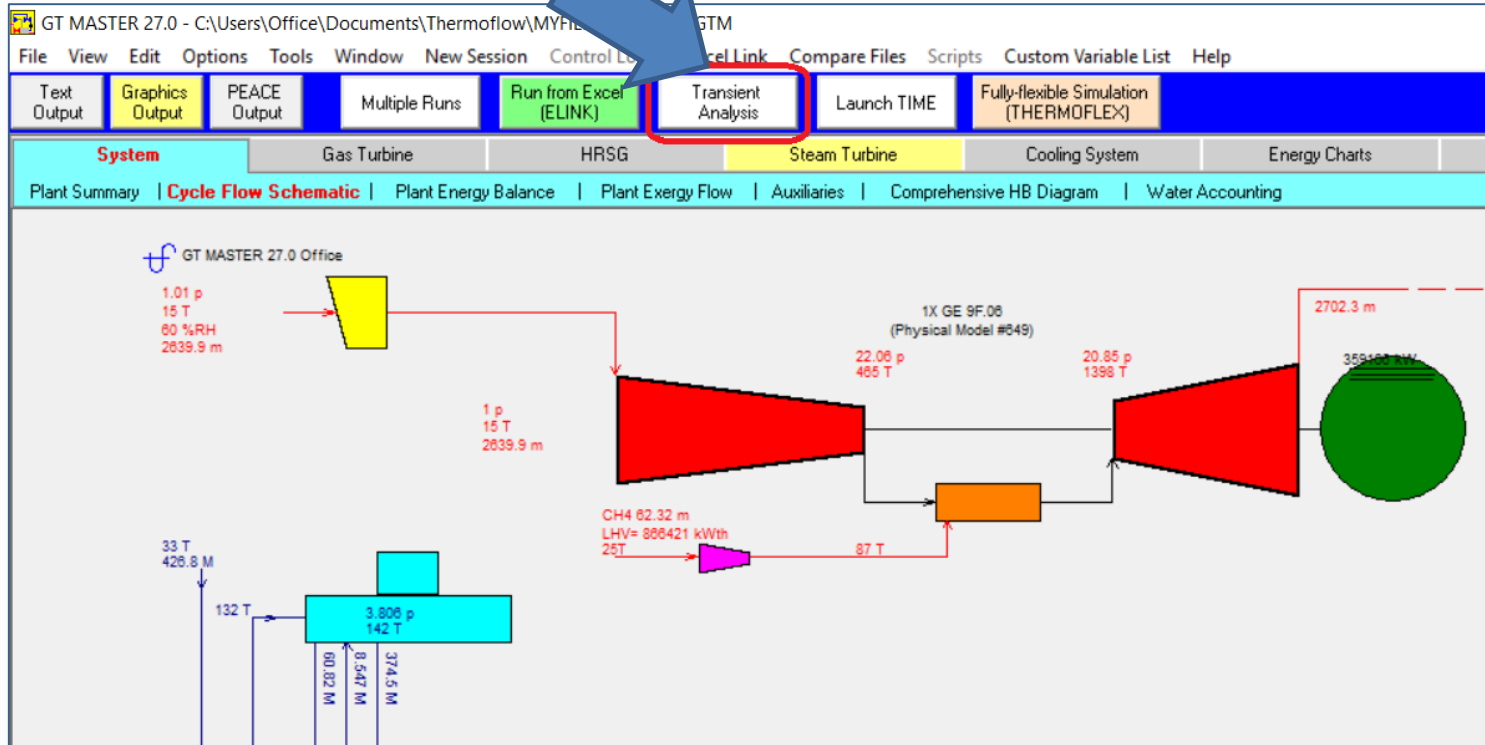
Steady State vs Transient Modelling

| | Steady state | Transient |
|--------------------------|---------------------|-----------------------|
| Time dependence | No | Yes |
| Stored energy simulated | No | Yes |
| Mass & energy in balance | Always in balance | Not always in balance |

General transient modelling application

- Determining plant operating flexibility.
- Developing control strategies for energy optimization purposes.
- Plant startup and shutdown.

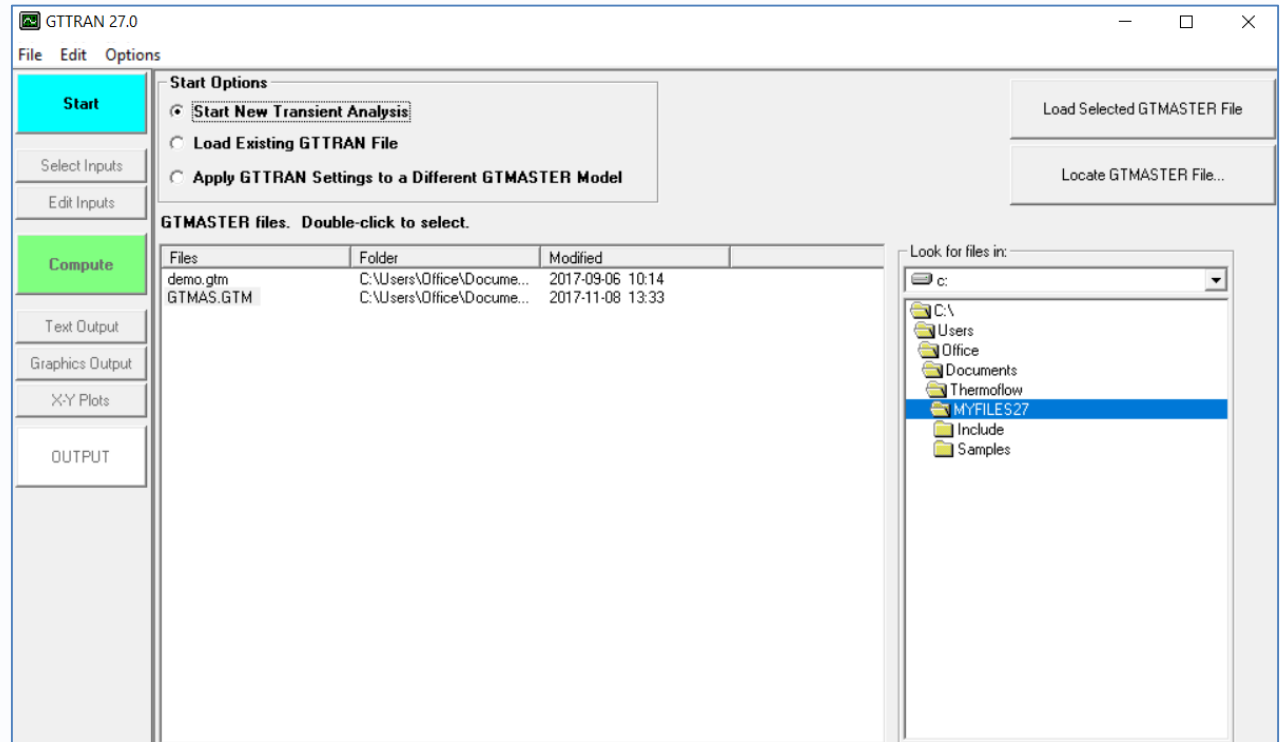
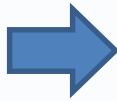
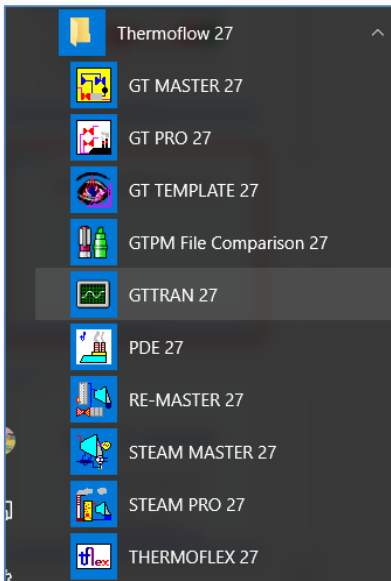
Launching GTTRAN from GT MASTER



“Transient Analysis” button is available in computed GT MASTER file.

Launching GTTRAN on its own

Windows Start /
All programs



Select Inputs

GTTRAN 27.0 - C:\Users\Office\Documents\Thermoflow\MYFILES27\GTMAS.TGTM

File Edit Options

Start

Increments
Number of time increments:

Select Inputs

Available INPUT Variables

Selected INPUT Variables

INPUTS - C:\Users\Office\Documents\Thermoflow\MYFILES27\GTMAS.GTM

- [-] Control Loop
- [-] Plant Criteria
- [-] GT Inputs
 - Number operating GT/HRSG
 - GT power as % of site rating**
 - GT generator output power (per GT)
 - Fuel line pressure
 - [-] User-defined Gas Turbine
 - [-] Source Gas Definition (No Gas Turbine Option)
 - Inlet filter pressure drop correction factor
 - Duct & Stack pressure drop correction factor
 - [-] Steam / Water Injection
 - [-] Fuel Definition
 - [-] Emissions
 - [-] Inlet Heating & Cooling
 - [-] Model Adjustments
 - [-] Controls, Margins, & Genset Losses
 - [-] Fuel Heating
 - [-] Bleeds & Additions
 - [-] Miscellaneous
 - [-] Chilled Water Storage 24-hr Model
- [-] ST Inputs
- [-] ST Process
- [-] HRSG Inputs
- [-] HRSG Process
- [-] Cooling System

51 GT power as % of site rating
829 Exhaust bypass
834 Duct burner exit temperature (db mode=0)

Undo Selections

Edit Inputs

GTTRAN 27.0 - C:\Users\Office\Documents\Thermodflow\MYFILES27\GTMAS.TGTM

File Edit Options

Start

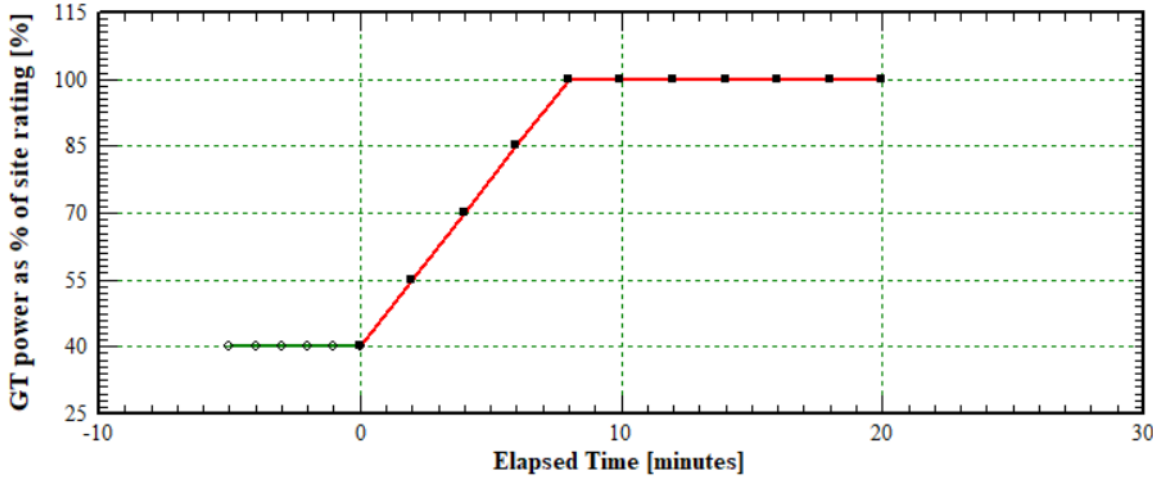
Transient Inputs Convergence Settings

Values for time step size, and model inputs can be entered directly on grid below, or by using the selection list and range entries to the right.

Time increment minutes @ case number
 to minutes @ case number

| | Case # | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 |
|--|---------|-------------------|--------|--------|--------|--------|---------|---------|---------|---------|---------|---------|
| Elapsed time | Minutes | Initial Condition | 2 min. | 4 min. | 6 min. | 8 min. | 10 min. | 12 min. | 14 min. | 16 min. | 18 min. | 20 min. |
| Time increment | Minutes | N/A | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 | 2 |
| GT power as % of site rating | % | 40 | 55 | 70 | 85 | 100 | 100 | 100 | 100 | 100 | 100 | 100 |
| Exhaust bypass | % | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 |
| Duct burner exit temperature (db mode=0) | C | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 | 0 |

GT power as % of site rating [%]



GT MASTER OUTPUT

Edit Input notes

- Time interval from 1 to 3 minute is suggested to provide meaningful results.
- Time increments smaller than one minute doesn't improve accuracy because of the model assumptions.
- Transient (dynamic) response of an operating combined cycle undergoing load changes is in the range of 10 to 30 minutes.
- Initial condition is a steady state GT MASTER run (it can be reset in GTTRAN input)

Outputs

The program computes energy storage in the following:

- Water/steam.
- Heat exchanger tubes.
- Metal fins.
- Drums.
- Boiler's headers.
- Boiler's liner/casing.

Outputs

For each time increment, GT MASTER iterates to find the **rate of change in stored energy**, Q_s , for each heat exchanger in the boiler. A time increment is converged when the following are satisfied:

$$SE(t+dt) = SE(t) + Q_s * dt$$

and

$$Q_g = Q_w + Q_s + Q_{loss}$$

where:

dt = Time increment,

$SE(t+dt)$ = Stored energy at time $t+dt$,

$SE(t)$ = Stored energy at time t ,

Q_s = Rate of increase in stored energy,

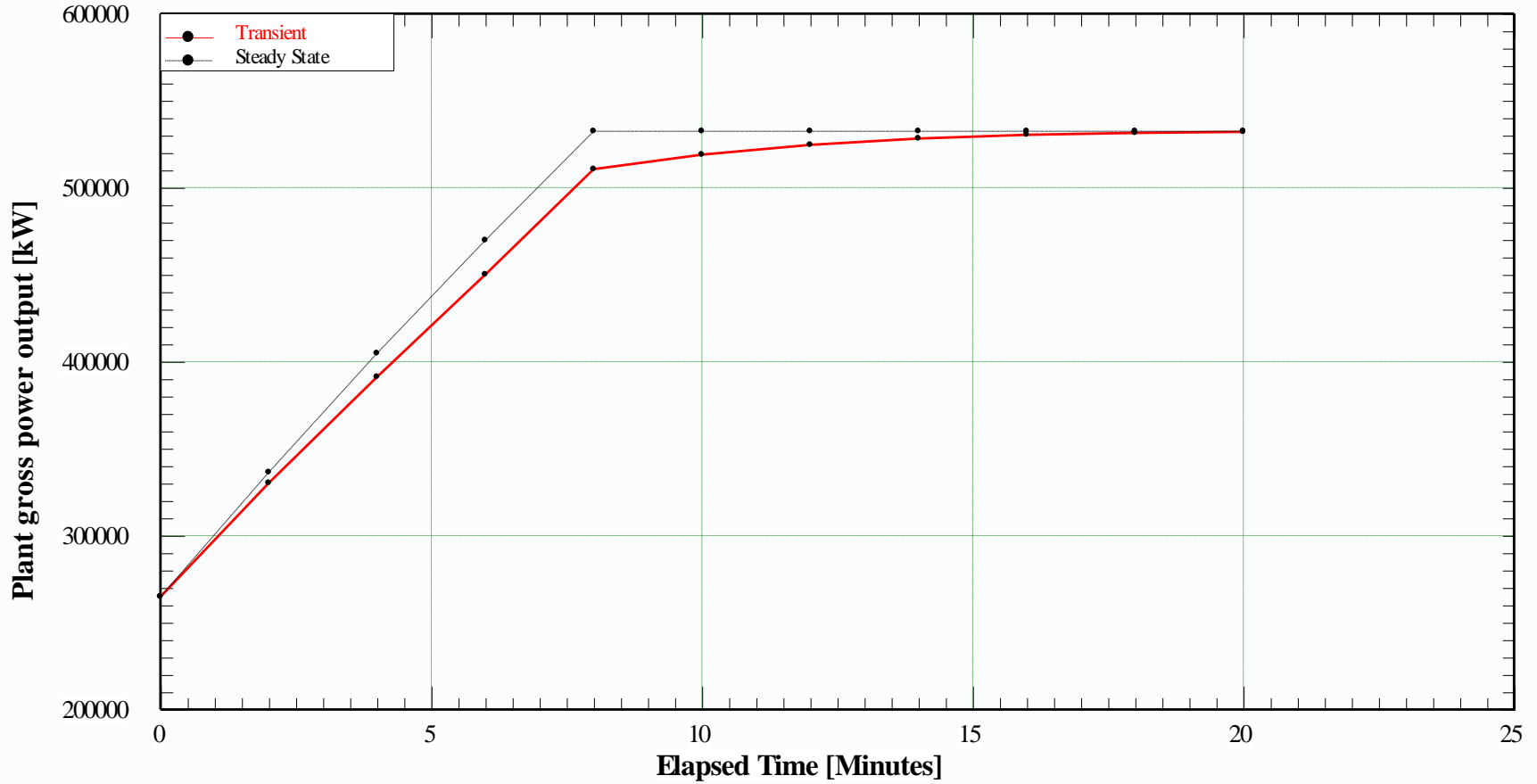
Q_g = Heat transfer rate from gas,

Q_w = Heat transfer rate to water/steam,

Q_{loss} = Rate of heat loss to the surroundings.

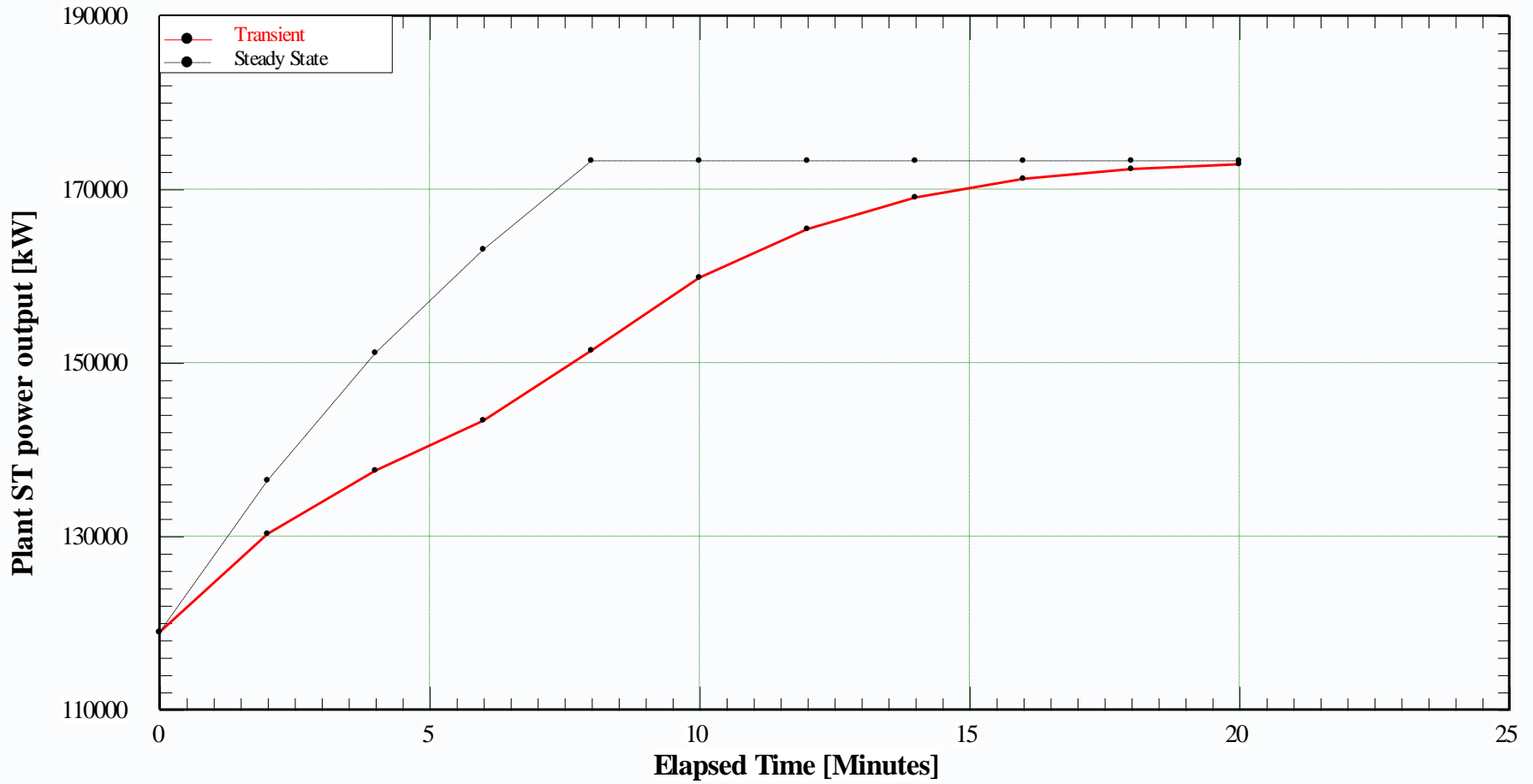
Outputs

Plant gross power output



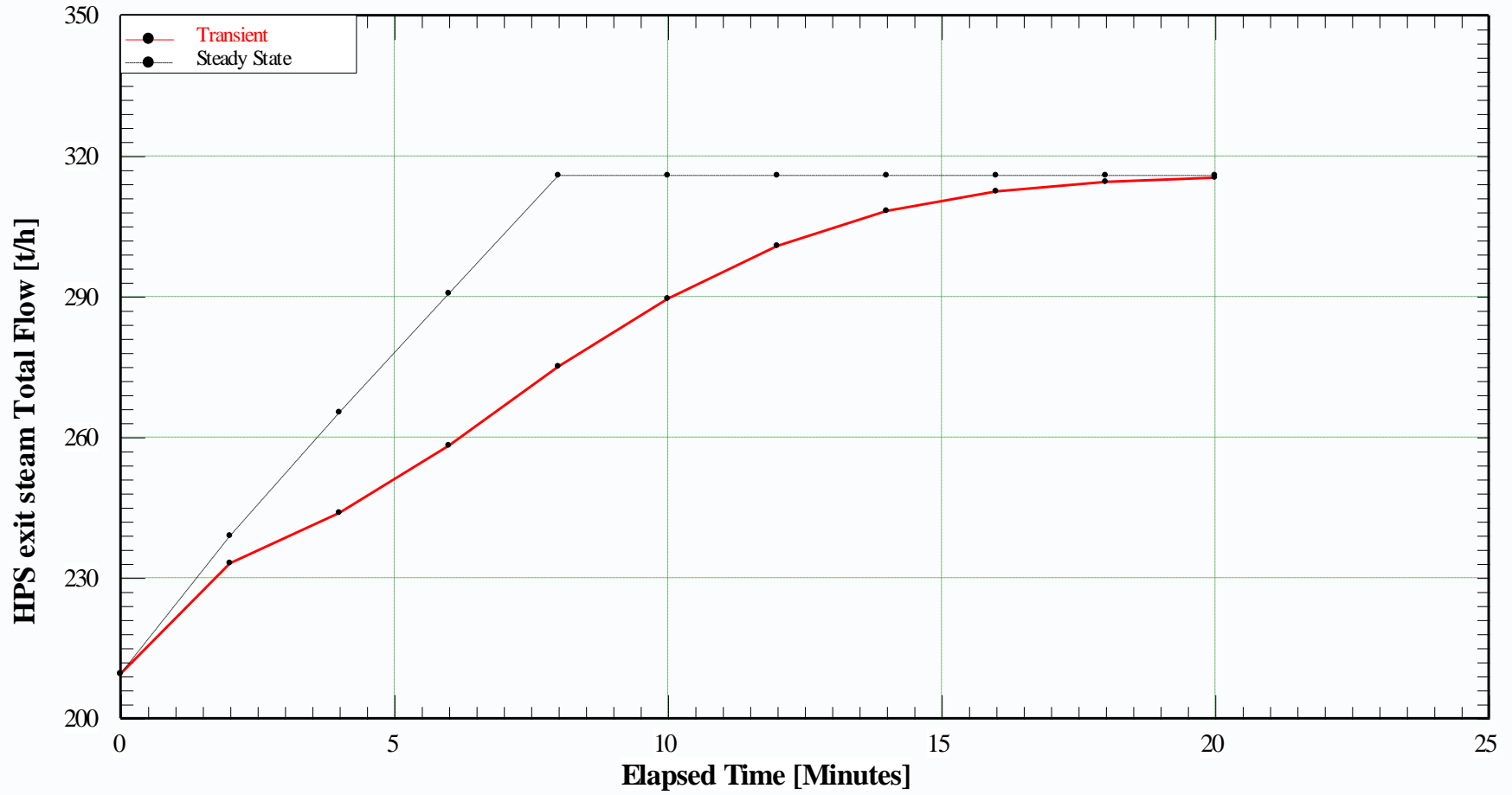
Outputs

Plant ST power output



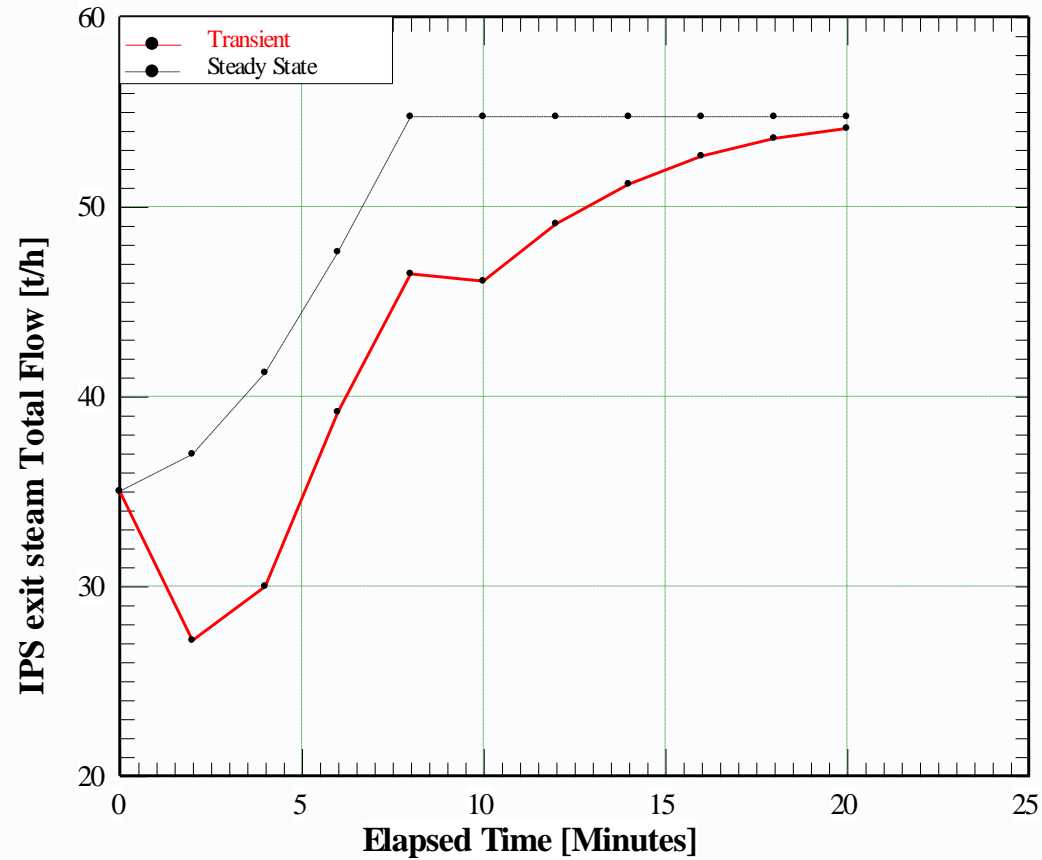
Outputs

HPS exit steam Total Flow

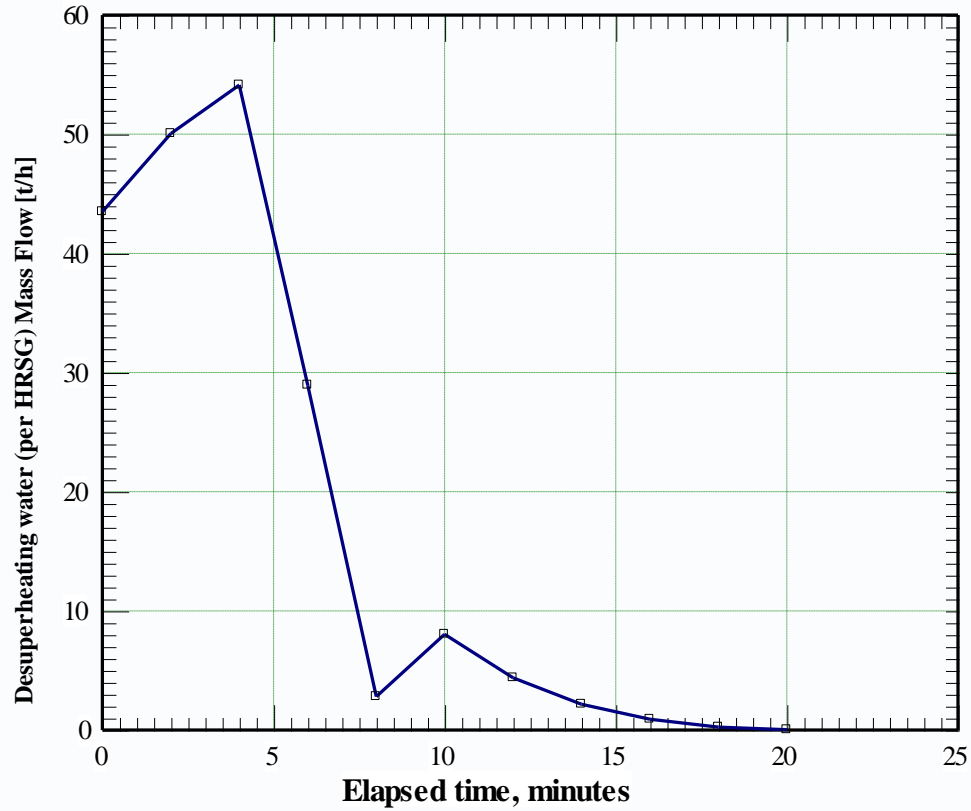


Outputs

IPS exit steam Total Flow



Outputs



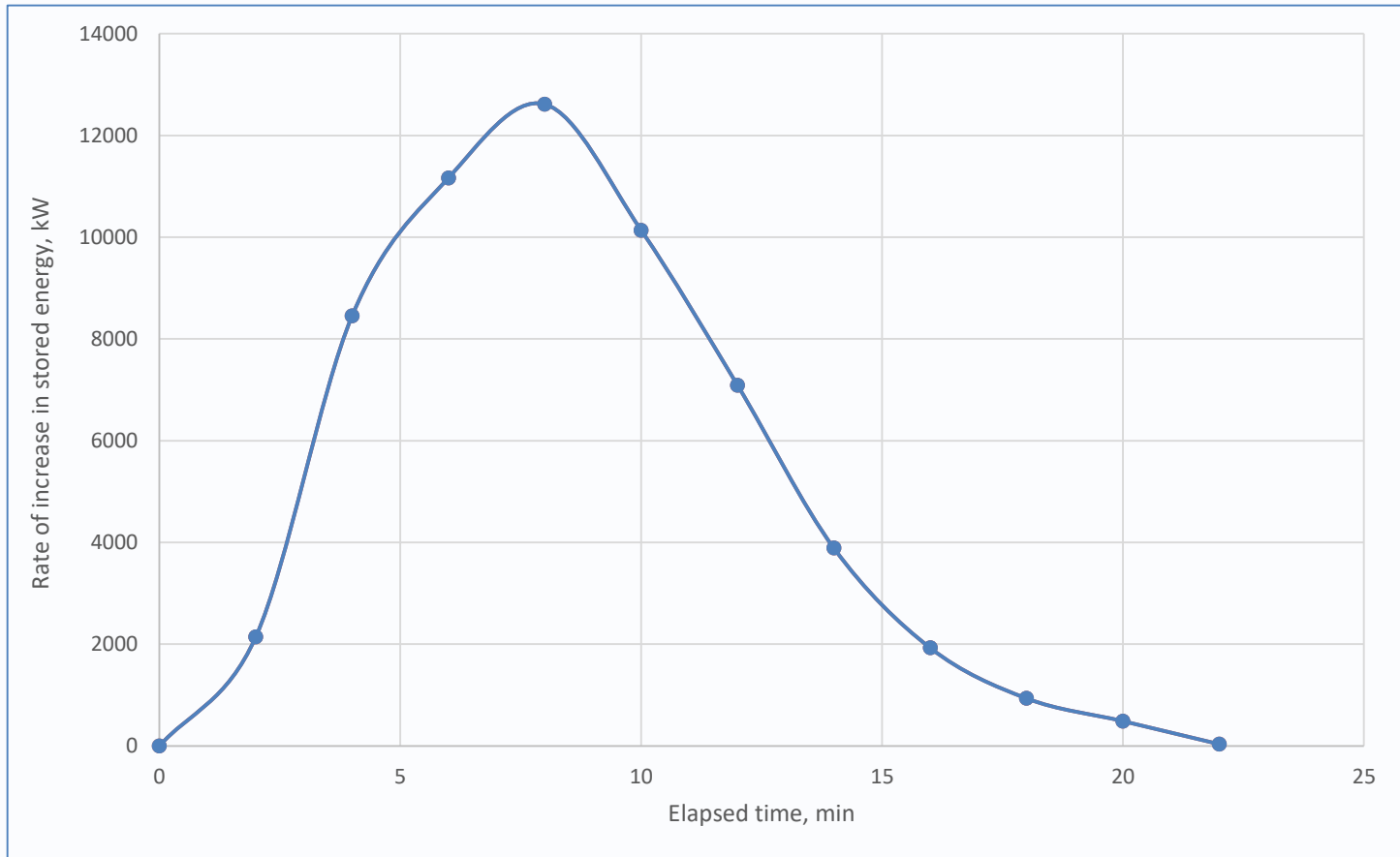
Outputs

HP Evaporator results

| Transient HRSG | Unit | 0 min | 2 min | 4 min | 6 min | 8 min | 10 min | 20 min | 22 min |
|--|-------------|--------------|--------------|--------------|--------------|--------------|---------------|---------------|---------------|
| GT load | % | 40 | 55 | 70 | 85 | 100 | 100 | 100 | 100 |
| Outputs | | HPB1 | HPB1 | HPB1 | HPB1 | HPB1 | HPB1 | HPB1 | HPB1 |
| 1. Heat transfer from gas | kW | 63241 | 72330 | 81163 | 89287 | 96141 | 95217 | 89528 | 89136 |
| 2. Heat transfer to water/steam | kW | 62926 | 69828 | 72303 | 77678 | 83048 | 84606 | 88596 | 88658 |
| 3. Rate of increase in stored energy | kW | 0 | 2142.6 | 8456 | 11165 | 12614 | 10137 | 487.1 | 34.26 |
| 4. Gas temperature entering HX | C | 447.8 | 449.1 | 452.1 | 454.6 | 456.7 | 459.3 | 458.4 | 458 |
| 5. Gas temperature exiting HX | C | 325.4 | 327.4 | 331.9 | 337.6 | 343.7 | 347.4 | 353.3 | 353.4 |
| 6. Water/steam temperature entering HX | C | 320.1 | 321.3 | 325.8 | 330.1 | 334.6 | 337 | 340.3 | 340.3 |
| 7. Water/steam temperature exiting HX | C | 319.9 | 320.6 | 323.7 | 328 | 332.8 | 336.7 | 343.3 | 343.4 |
| 8. Total fin weight | kg | 106105 | 106105 | 106105 | 106105 | 106105 | 106105 | 106105 | 106105 |
| 9. Average fin temperature | C | 331.4 | 332.9 | 337 | 341.9 | 347.3 | 351.1 | 356.7 | 356.8 |
| 10. Total tube weight | kg | 102408 | 102408 | 102408 | 102408 | 102408 | 102408 | 102408 | 102408 |
| 11. Average tube temperature | C | 319.9 | 320.6 | 323.7 | 328 | 332.8 | 336.7 | 343.3 | 343.4 |
| 12. Stored energy in fins | MJ | 20297 | 20414 | 20721 | 21106 | 21523 | 21815 | 22261 | 22266 |
| 13. Stored energy in tubes | MJ | 18750 | 18795 | 19024 | 19338 | 19695 | 19982 | 20469 | 20477 |
| 14. Stored energy in HX water/steam | MJ | 11487 | 11514 | 11652 | 11847 | 12076 | 12265 | 12601 | 12608 |
| 15. Stored energy in headers | MJ | 2503.9 | 2509.8 | 2540.1 | 2582.4 | 2631.2 | 2671.2 | 2740.7 | 2742.1 |
| 16. Stored energy in drums | MJ | 27457 | 27513 | 27799 | 28194 | 28643 | 29004 | 29618 | 29630 |
| 17. Stored energy in liner/casing | MJ | 1392.4 | 1398.8 | 1413.3 | 1429.4 | 1445.3 | 1457.6 | 1467.3 | 1466.7 |
| 18. Total stored energy | MJ | 81888 | 82144 | 83150 | 84497 | 86014 | 87195 | 89157 | 89190 |

Outputs

HP Evaporator results



Q & A session

Please send your questions to the
presenter in the webinar chat!

For further questions:
zakharenkov@thermoflow.com

Thank you!